INDOOR ENVIRONMENT-

New technology and sensors. "Setting the standard"

Professor Bjarne W. Olesen, PhD Director International Center for Indoor Environment and Energy Department of Mechanical Engineering Technical University of Denmark





INDOOR - OUTDOOR

- Highest exposure to the indoor environment
- People spend ~90 % of the time indoors during work, during transportation and at home

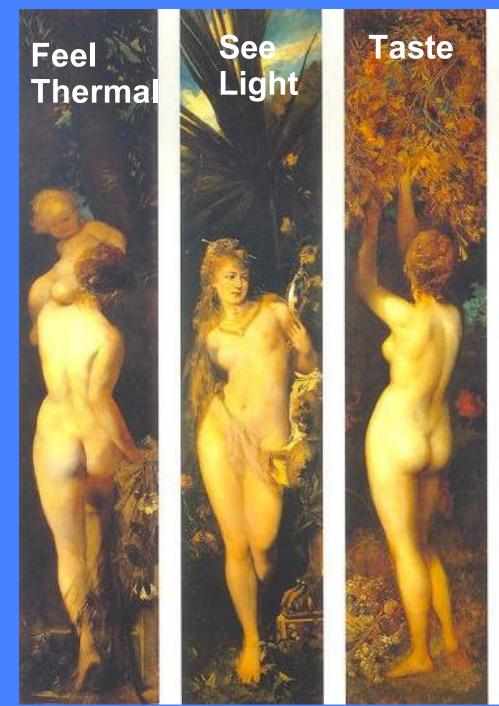
INTAKE FOR A PERSON PER DAY

- 1 kg FOOD
- 2 kg LIQUID
- 15 kg AIR

COMFORT-PRODUCTIVITY Building costs

People100Maintenance10Financing10Energy1





Hear Acoustic





INDOOR ENVIRONMENT

- THERMAL
- AIR QUALITY
- ACOUSTIC
- LIGHT

Large field studies show substantial rates of dissatisfaction in practice

(Mendell, 1993; Fisk et al., 1993; Bluyssen et al., 1994; Sundell, 1994; Sekhar et al., 2000; Bischhof 2000)

In typical office buildings

20-40% of occupants with SBS symptoms

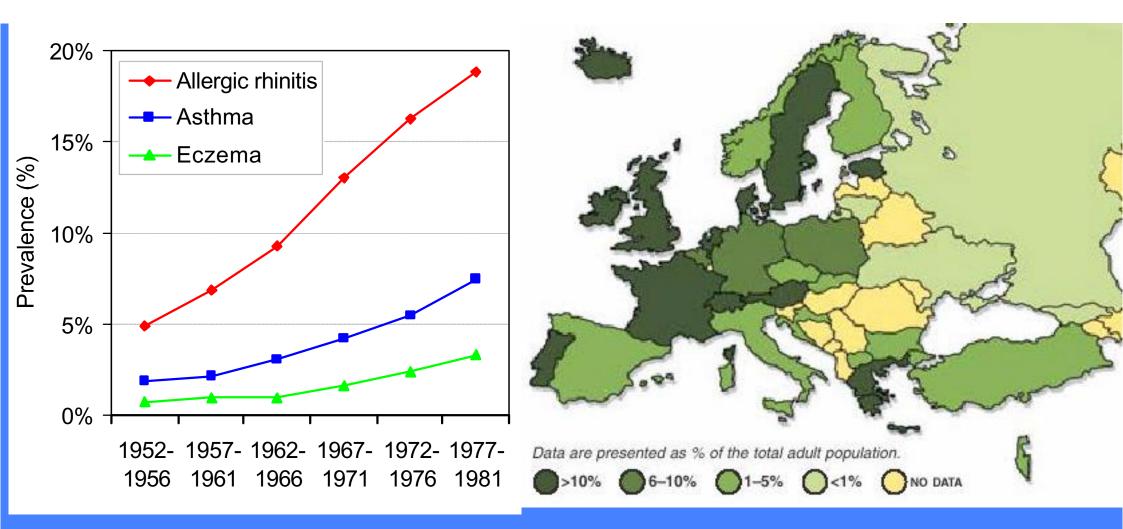
20-40% of occupants finding the IAQ unacceptable

even though existing ventilation standards are met.

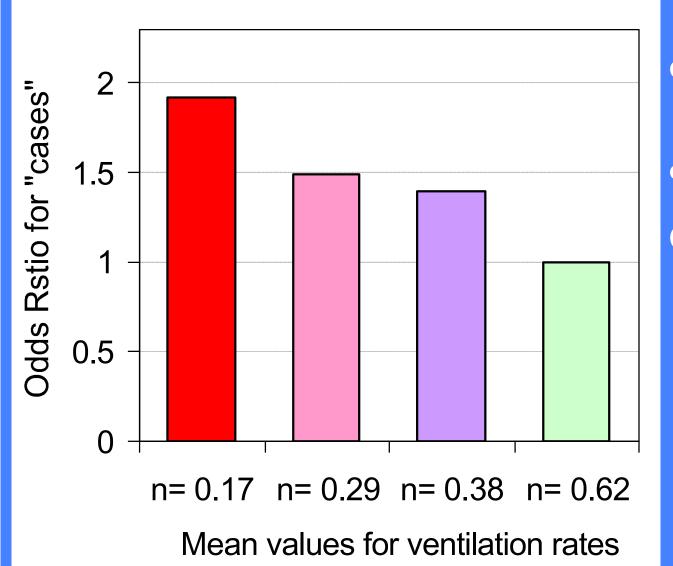
We guarantee that less than 30% of our customers will be dissatisfied !





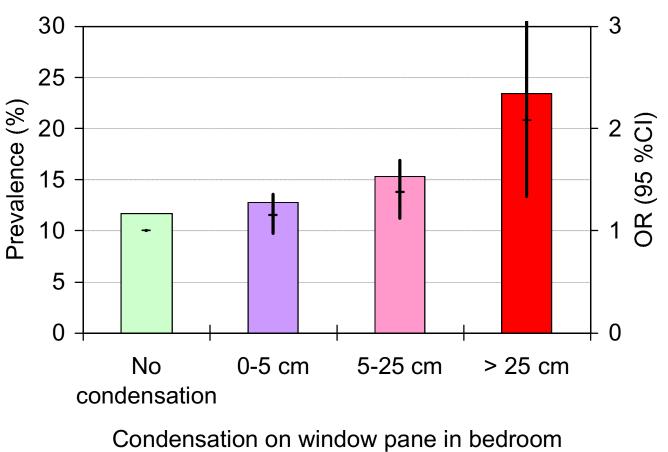


<u>Left:</u> Trends for allergic rhinitis, asthma and eczema among male conscripts (17-20 years age) in Sweden (Bråbäck et al., 2004). <u>Right:</u> Current data on prevalence of asthma in adults in Europe (Loddenkemper et al., 2003).



Odds ratio for being a "case", i.e. children with at least two symptoms of possible three (wheezing, rhinitis, eczema) as a function of ventilation rates, in single family houses. (Bornehag et al., 2003).

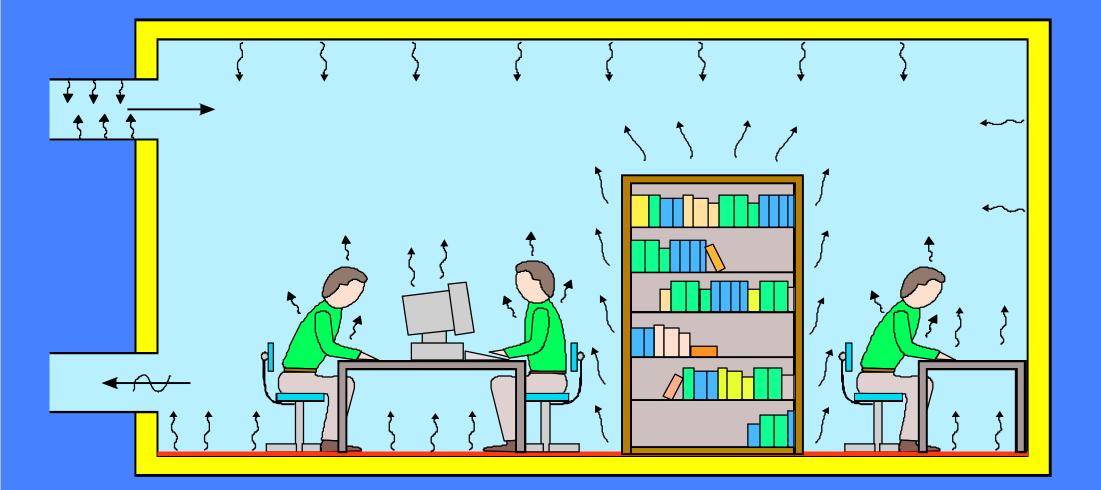




Left: Window water condensation is often a sign of poor ventilation in dwellings;

<u>*Right: Prevalence and odds ratio for rhinitis among children versus condensation on window pane in a bedroom (source: DBH-study group, in press).</u></u>*

Indoor Pollution Sources



List of 350 common volatile organic compounds in indoor air

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C.Preditors

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M. PENTHES

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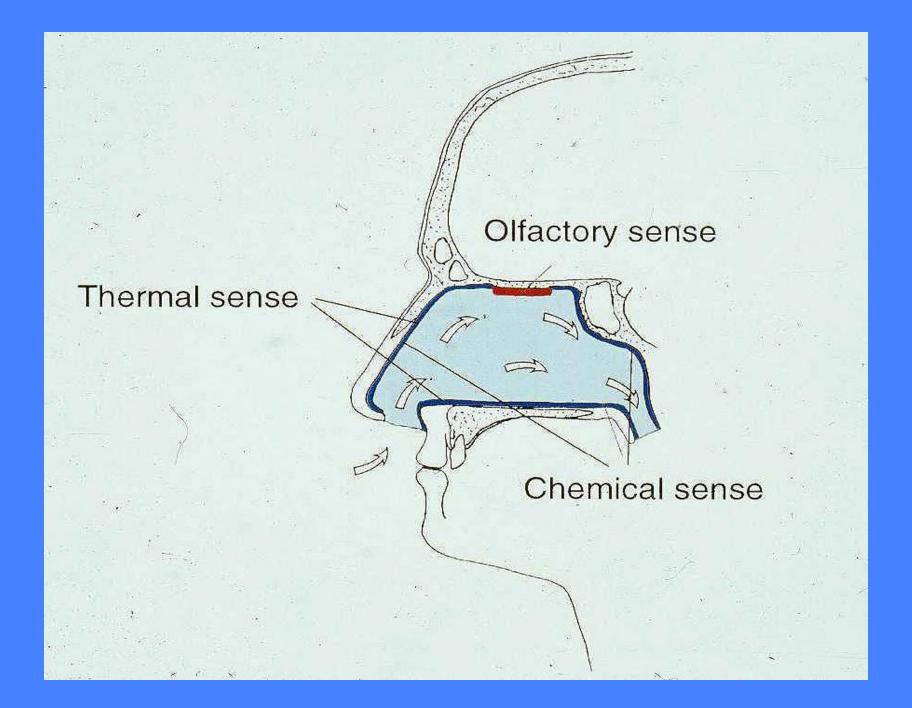
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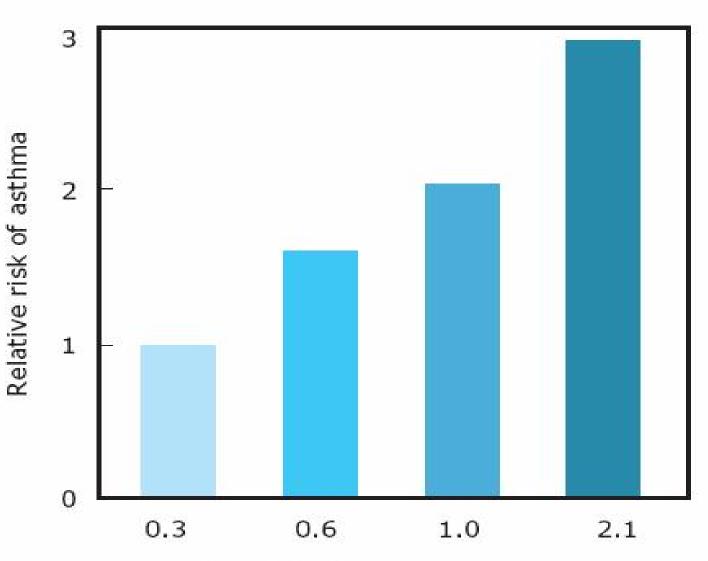
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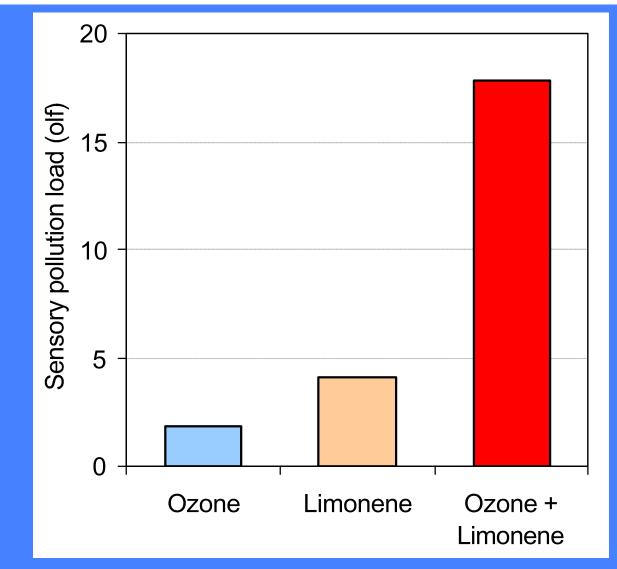


Plasticizers from polyvinyl chloride in dwellings increase the risk of asthma among children.

Each column represents about 90 dwellings.

Median phthalate concentration: mg of DEHP per gram of dust in the home

DEHP: di(2-ethylhexyl) phthalate.



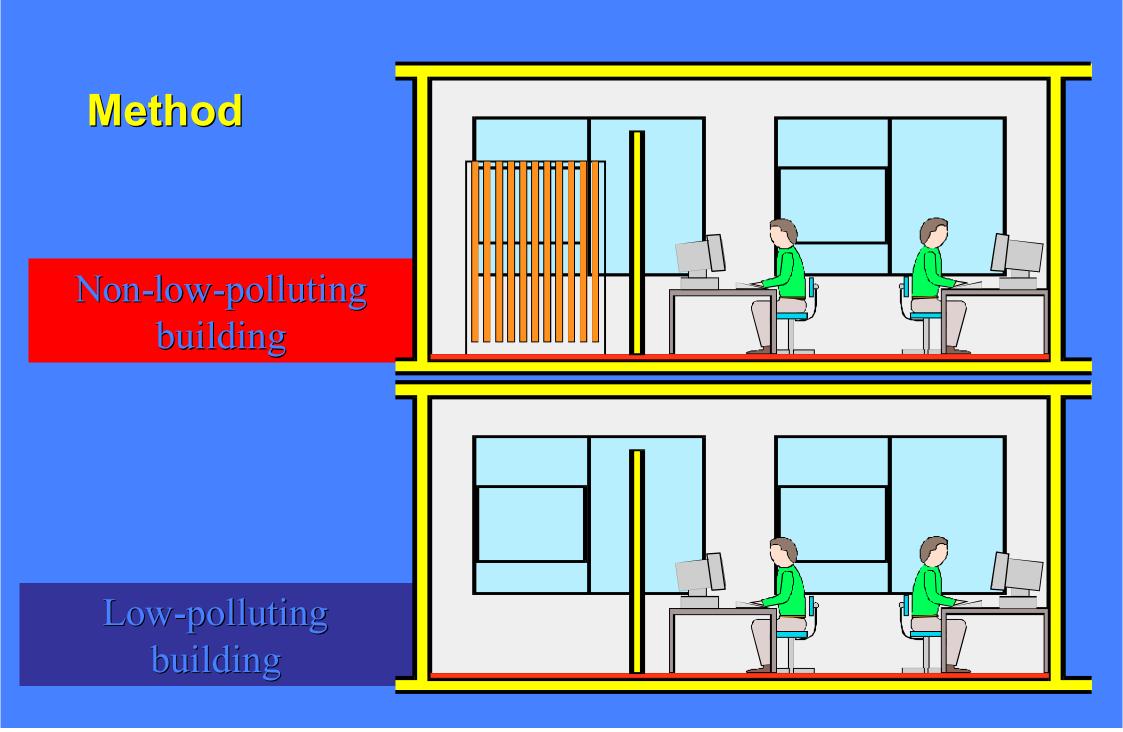
The sensory pollution load in an office where either ozone (15 ppb) or limonene (83 ppb) were present separately or both ozone and limonene (15 ppb+83 ppb respectively) were mixed in the office air (Tamás et al., 2005); the increased sensory pollution load is due to the presence of reaction products in the office air.

INDOOR ENVIRONMENT-PERFORMANCE

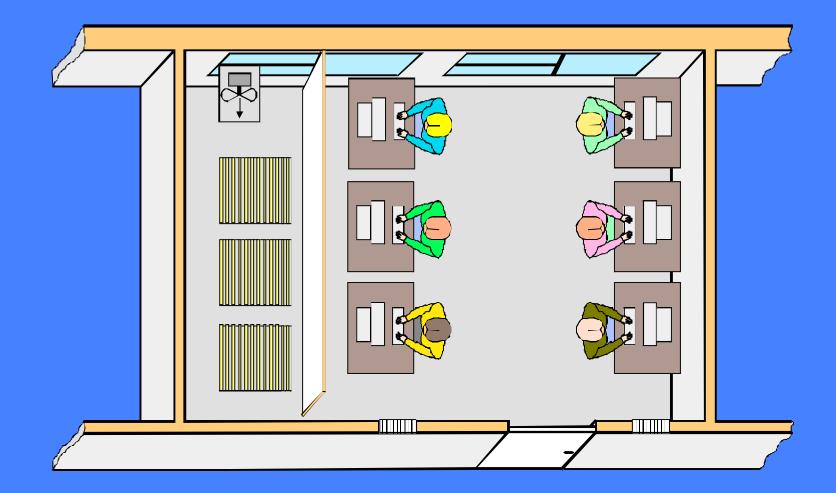
- Self evaluation of productivity-WEP
 9-Punkt scale
- WEP = 6.739-0.419E-0.164JD-0.048JS
- E = Dissatisfaction-Environment
- JD = Job satisfaction
- JS = Job stress

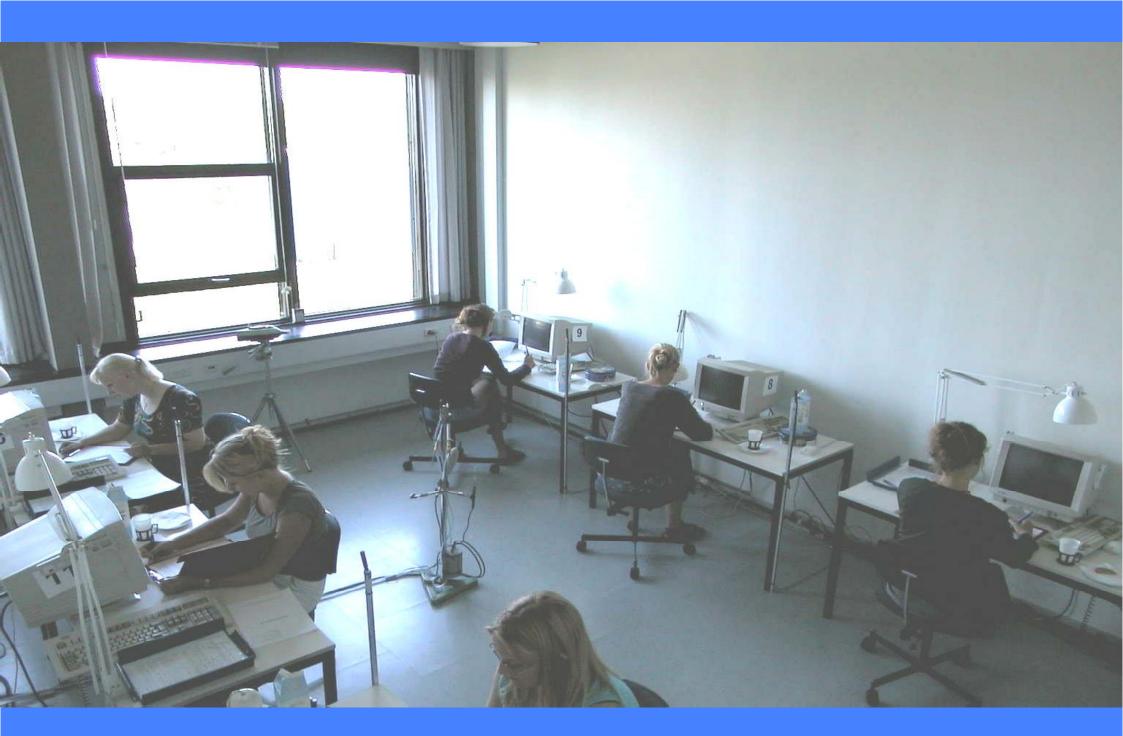
PERFORMANCE

Studies on the relation between the indoor environment and peoples performance



Experimental set-up



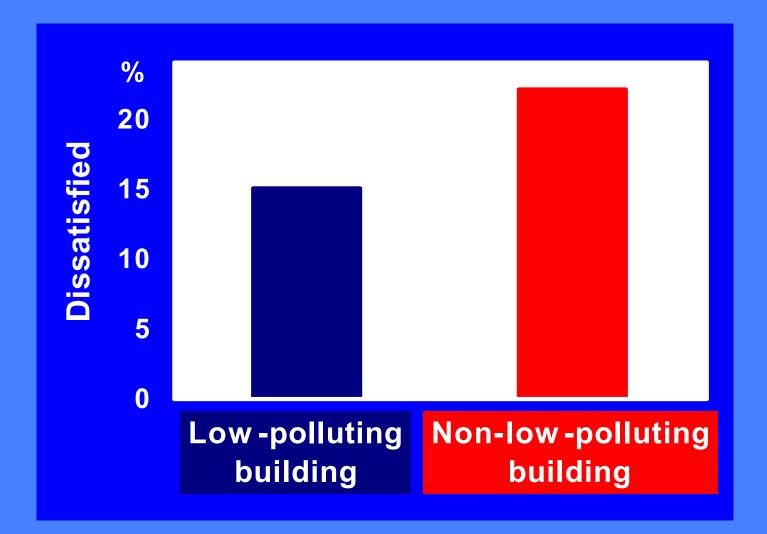


Subjects & procedure

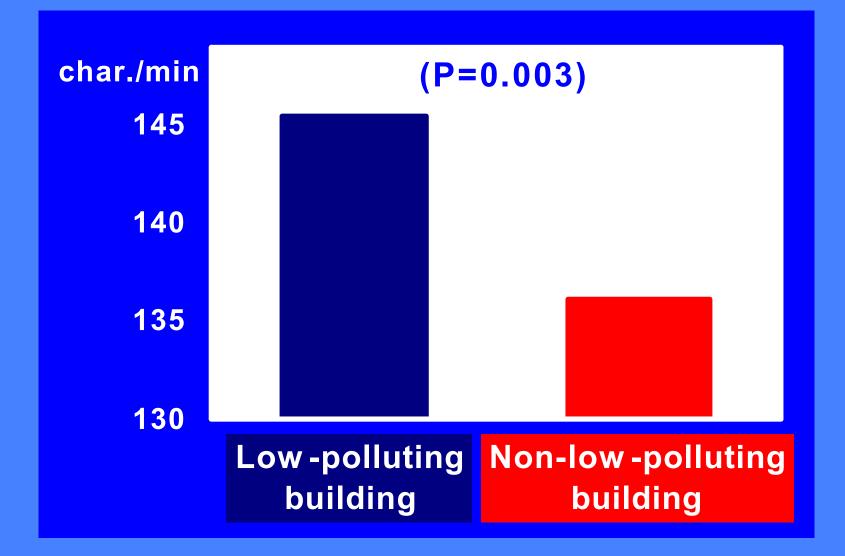
- 30 healthy females
- 20-31 years old
- performed simulated office work durin
 4.4 hours' occupation of the office:
 - text typing
 - arithmetical calculations
 - creative thinking



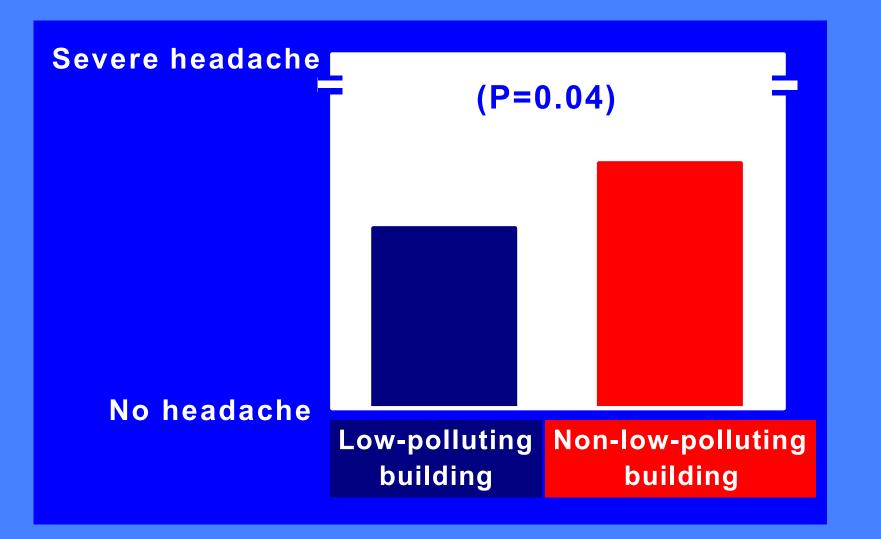
Perceived air quality



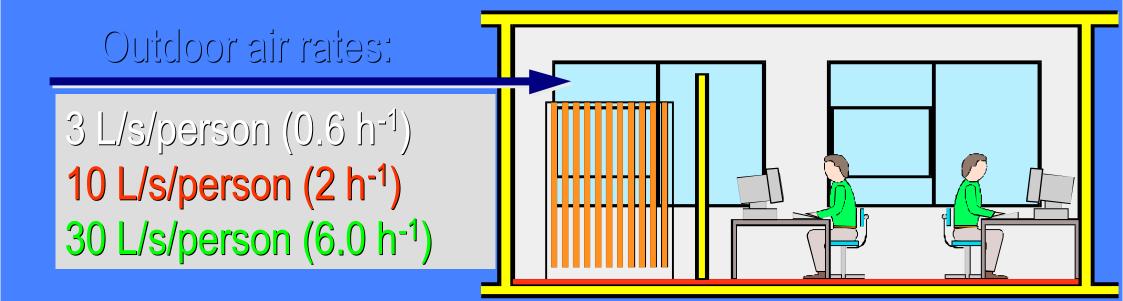
Speed of text typing



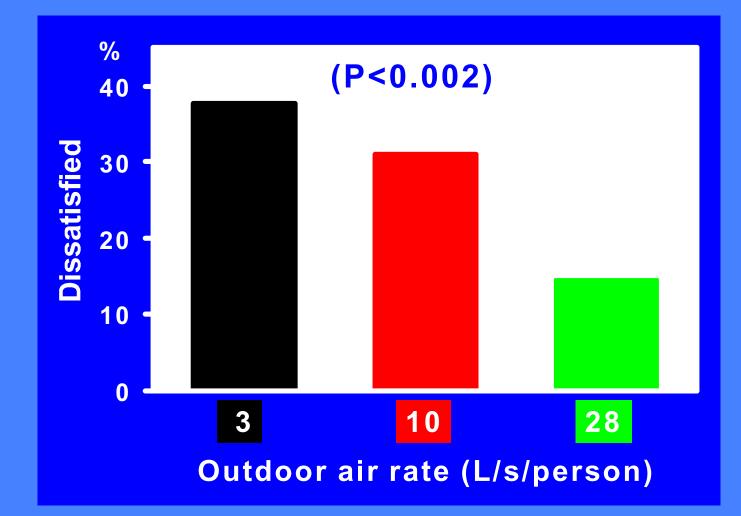
Headache



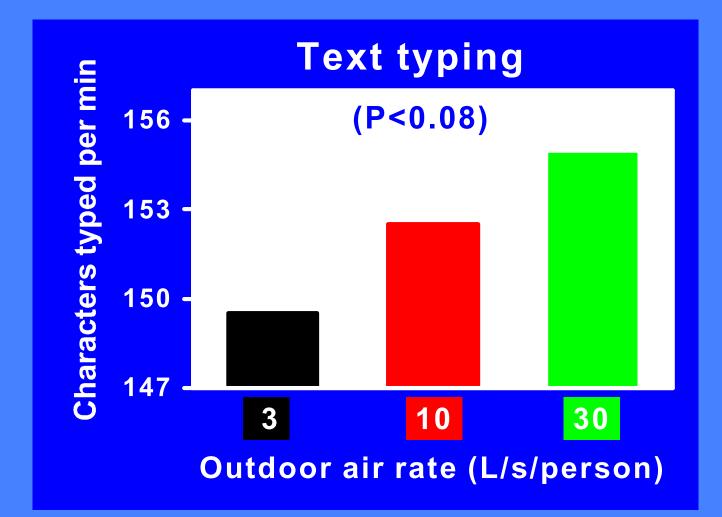
Methods



Perceived air quality



Productivity



The results of these 3 studies were confirmed in actual workplace: a call-centre

Performance measure



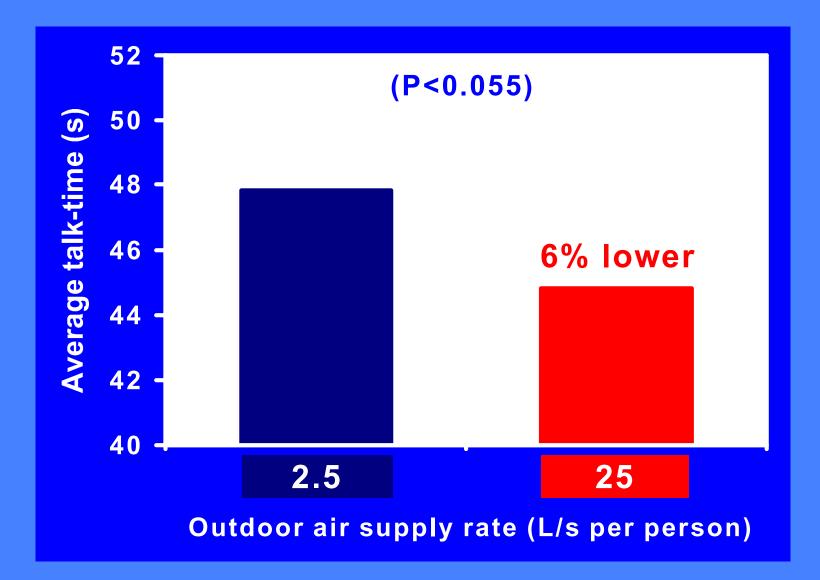
Average

time

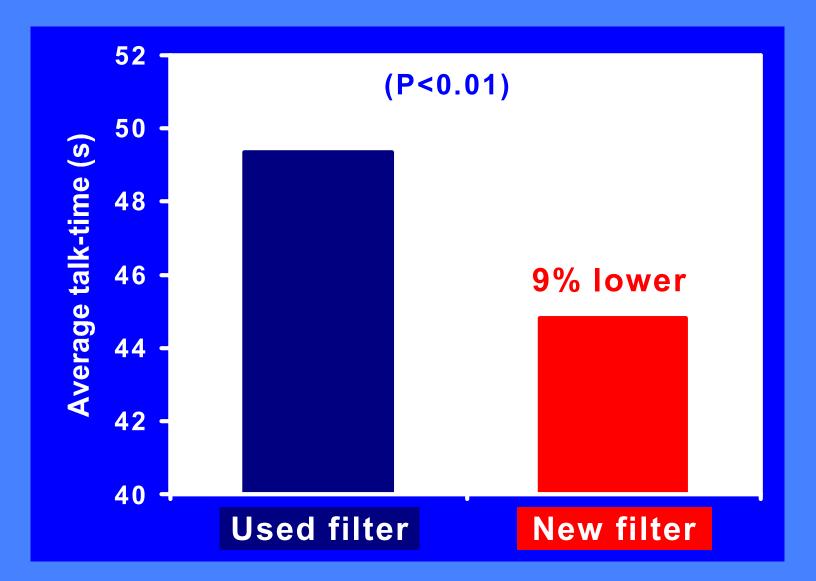
talk

Number

Average talk-time with NEW FILTER

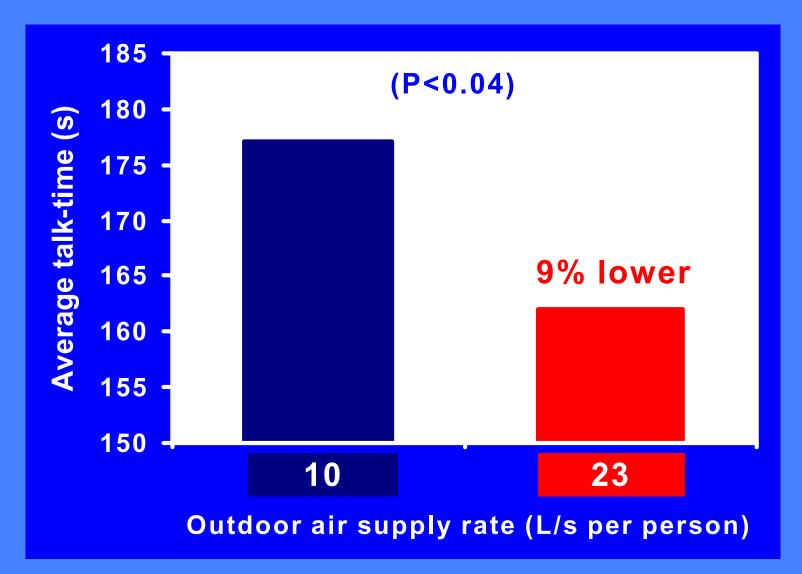


Average talk-time with HIGH OUTDOOR AIR RATE

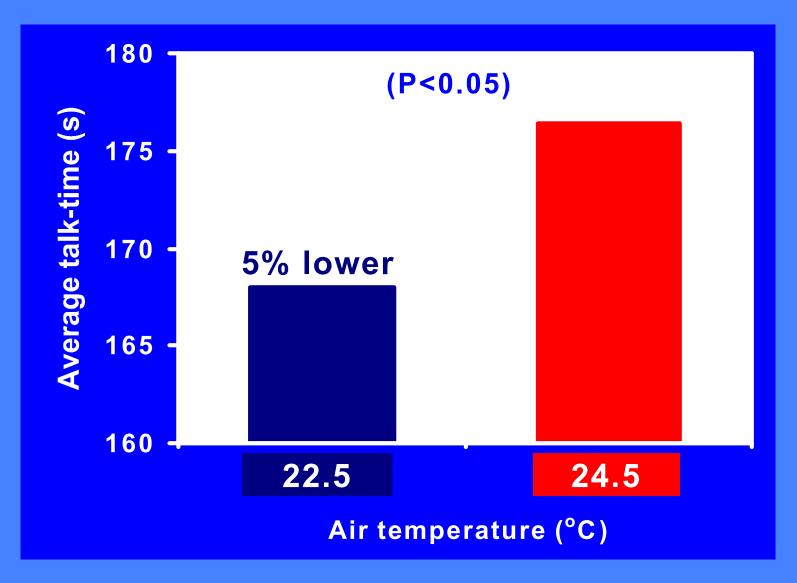


This study has been repeated in another country (Singapore) with similar results

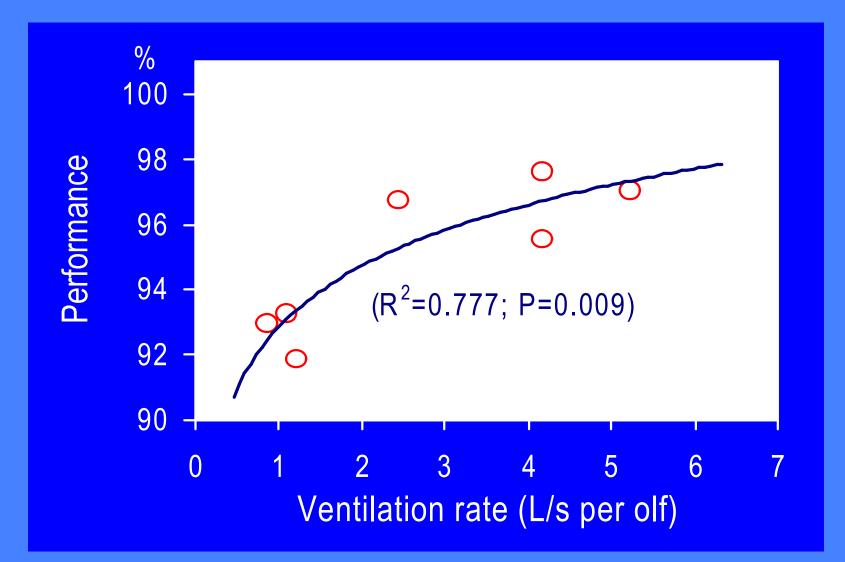
Average talk-time at AIR TEMPERATURE 24.5°C



Average talk-time with LOW OUTDOOR AIR RATE



Ventilation vs performance

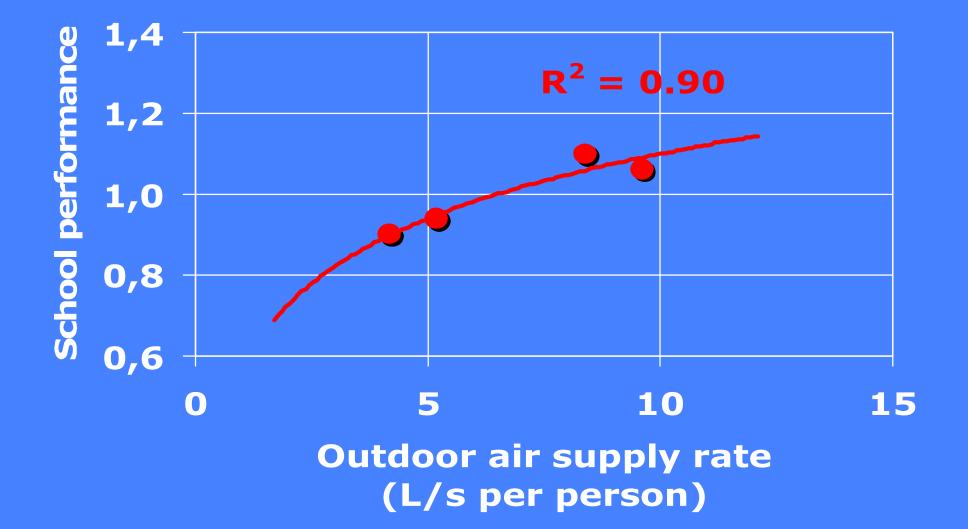


5 elementary schools



Children at an age of 10-12 years old (4-6 grade)

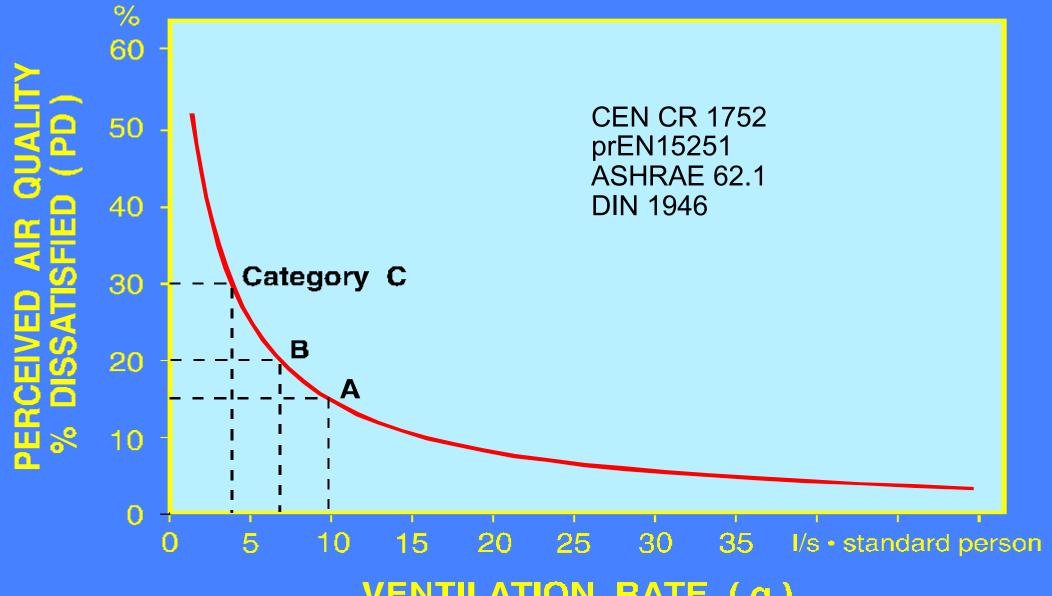
School performance and ventilation Pilot study- ASHRAE project



Concept for calculation of design ventilation rate

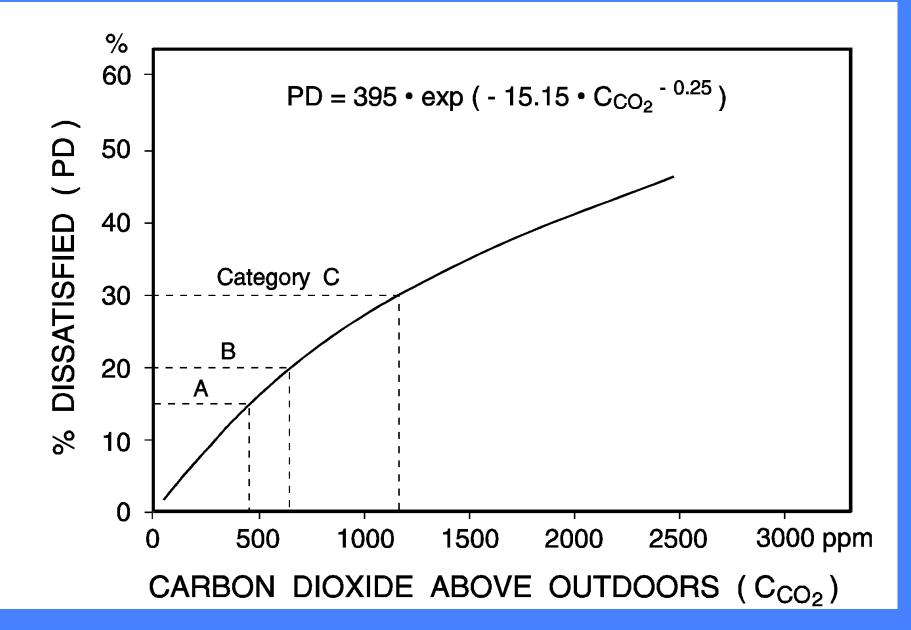
People Component Building Component Breathing Zone Outdoor Airflow R_sS $\mathbf{R}_{\mathbf{p}}\mathbf{P}_{\mathbf{z}}$ R_aA_z V_{bz} +____ Minimum Ventilation **Building Area** I/s/Person per Smoke Number of Number of **Minimum** People **Smokers** l/s/m²





VENTILATION RATE (q)





Generation of pollution from occupants

	Sensory pollution load olf/ occupant	Carbon dioxide I/(h [.] occupant)	Carbon monoxide ¹⁾ I/(h⋅ occupant)	Water vapour ²⁾ g/(h· occupant)
Sedentary, 1-1.2 met				
0% smokers	1	19		50
20% smokers ³⁾	2	19	11·10 ⁻³	50
40% smokers ³⁾	3	19	21 ·10 ⁻³	50
Physical exercise				
Low level, 3 met	4	50		200
Medium level, 6 met	10	100		430
high level (athletes), 10 met	20	170		750
Children				
Kindergarten, 3-6 years, 2,7	1,2	18		90
met	1,3	19		50
School, 14-16 years, 1-1,2 met	\downarrow			_

¹⁾ from tobacco smoking

²⁾ applies for persons close to thermal neutrality

³⁾ average smoking rate 1,2 cigarettes/hour per smoker, emission rate 44 ml CO/cigarette

POLLUTANT FROM BUILDING, FURNISHING AND SYSTEM	Sensory pollution load olf/m ²		
	Average	Range	
Existing buildings			
Offices ^{a)}	0,3 ^{d)}	0,02-0,95	
Offices ^{b)}	0,6 °)	0 - 3	
Class rooms ^{a)}	0,3	0,12-0,54	
Kindergarten ^{a)}	0,4	0,20-0,74	
Auditorium ^{a)}	0.3 ^{d)}	0,13-1,32	
New buildings (No-smoking)			
Low polluting buildings	0,1		
Not-low polluting buildings	0,2		
Very low polluting buildings	0.02		
 a) Data from more than 40 mechanical ventilated buildings in b) Data from an European Audit program, 1992-1995. c) Includes pollutant load from smokers d) Includes pollutant from earlier smoking. 	n Denmark.		

Type of building/ space	Occu- pancy person/m ²	Cate- gory CEN	OccupantsAdditional ventilation foronlybuilding (add only one)l/s personl/s · m²		Total l/s∙m ²				
			ASH- RAE Rp	CEN	CEN low- polluting building	CEN <i>Non-</i> low- polluting building	ASH- RAE Ra	CEN Low Pol.	ASH- RAE
Single office (cellular office)	0,1	A B C	2,5	10 7 4	1,0 0,7 0,4	2,0 1,4 0,8	0,3	2 1,4 0,8	0,55
Land- scaped office	0,07	A B C	2,5	10 7 4	1,0 0,7 0,4	2,0 1,4 0,8	0,3	1,7 1,2 0,7	0,48
Confe- rence room	0,5	A B C	2,5	10 7 4	1,0 0,7 0,4	2,0 1,4 0,8	0,3	6 4,2 2,4	1,55

 $1 \text{ l/s } \text{m}^2 = 0.2 \text{ cfm/ft}^2$

MODERATE THERMAL ENVIRONMENT

ISO EN 7730-2005

 Ergonomics of the thermal environment – Analytical determination and interpretation of thermal comfort using calculation of the PMV and PPD indices and local thermal comfort effects.

ASHRAE 55-2004

Thermal environment conditions for human occupancy

• CR 1752

Ventilation of buildings-Design criteria for the indoor environment

prEN15251 (European Energy Performance of Buildings Directive)

- Criteria for the indoor environmental quality

• EN 13779

Ventilation for non-residential buildings - performance requirements for ventilation and room-conditioning systems

TC204WG5 Building Environmental Design

Thermal Environment



MODERATE ENVIRONMENTS

• GENERAL THERMAL COMFORT – PMV / PPD, OPERATIVE TEMPERATURE

LOCAL THERMAL DISCOMFORT

- Radiant temperature asymmetry
- Draught
- Vertical air temperature difference
- Floor surface temperature



GENERAL THERMAL COMFORT

- Personal factors
 - Clothing
 - Activity
- Environmental factors
 - Air temperature
 - Mean radiant temperature
 - Air velocity
 - Humidity

 $PMV = (0,303e^{-0,036M} + 0,028)$

$$\begin{bmatrix} (M - W) - 3,05 \cdot 10^{-3} \{ 5733 - 6,99(M - W) - p_a \} - 0,42 \{ (M - W) - 58,15 \} \\ -1,7 \cdot 10^{-5} M (5867 - p_a) - 0,0014 M (34 - t_a) \\ -3,96 \cdot 10^{-8} f_{cl} \{ (t_{cl} + 273)^4 - (\bar{t}_r + 273)^4 \} - f_{cl} h_c (t_{cl} - t_a)$$

where

$$t_{cl} = 35,7 - 0,028(M - W) - I_{cl} \left[3,96 \cdot 10^{-8} f_{cl} \left\{ (t_{cl} + 273)^4 - (\bar{t}_r + 273)^4 + f_{cl} h_{cl} (t_{cl} - t_a) \right\} \right]$$

$$\begin{split} h_{c} &= 2,38 (t_{cl} - t_{a})^{0.25} \text{ for } 2,38 (t_{cl} - t_{a})^{0.25} > 12,1 \sqrt{v_{ar}} \\ &= 12,1 \sqrt{v_{ar}} \quad \text{for } 2,38 (t_{cl} - t_{a})^{0.25} < 12,1 \sqrt{v_{ar}} \\ f_{cl} &= \frac{1,00 + 1,290 I_{cl}}{1,05 + 0,545 I_{cl}} \dots \text{ for } I_{cl} < 0,078 m^{2} \cdot KW^{-1} \\ &= \int_{cl}^{1,00 + 1,290 I_{cl}} \frac{1}{1,05 + 0,545 I_{cl}} \dots \text{ for } I_{cl} > 0,078 m^{2} \cdot KW^{-1} \end{split}$$

PMV

	_	Fieulcieu Meall Volg
$M \\ W$	=	Metabolic rate Wm^{-2}
I_{cl}	=	Effective mechanica (100 Wm^{-2})
f_{cl}	=	Clothing insulation
t 5 01	=	Clothing area factor

Pradicted Moon Vote

- Air temperature, °C =
- t_a \overline{t}_r v_{ar} =
- Mean radiant temperature, °C Relative air velocity, ms^{-1} P_a p_a =
- h_c Water vapour particle pressure, = $W \cdot m^{-2} K^{-1}$
- Convective heat transfer coefficient, t_{cl} =
 - Clothing surface temperature, °C =

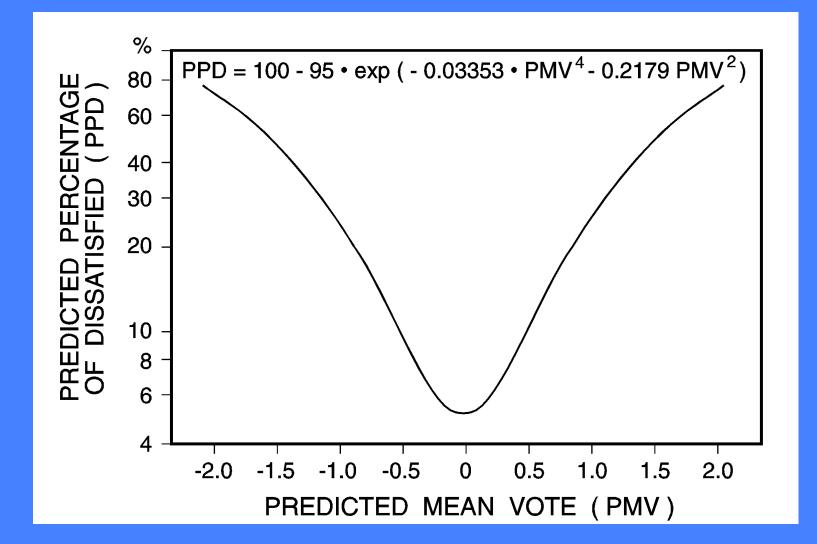
PMV-index

PMV-index

- -3 Cold
- -2 Cool
- -1 Slightly cool
- 0 Neutral
- +1 Slightly warm
- +2 Warm
- +3 Hot



GENERAL THERMAL COMFORT





2,2 met

21°C

1 clo

1,6 met

20°C







0,8 clo 1,6 met 21°C

0,8 clo

1,6 met

21°C



GENERAL THERMAL COMFORT

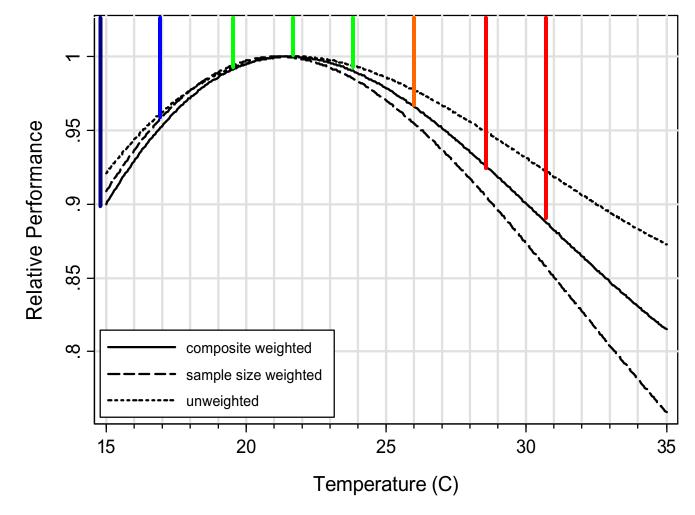
Category	Thermal state of the body as a whole		
	PPD	Predicted Mean Vote	
	%		
Α	< 6	-0.2 < PMV < +0.2	
В	< 10	-0.5 < PMV < +0.5	
С	< 15	0.7 < PMV < +0.7	

THERMAL COMFORT

- OPERATIVE TEMPERATURE
- -0,5 < PMV < +0,5 ; PPD < 10 %
- SPACES WITH MAINLY SEDENTARY OCCUPANTS :
 - SUMMER CLOTHING 0,5 clo
 - ACTIVITY LEVEL 1,2 met
- 23 °C < t_o < 26 °C.

PMV-values

<u>-1.5 -1.0 -0.5 0 0.5 1.0 1.5 2.0</u>



from Seppänen and Fisk 2005a

PMV-value	PPD- Value %	Reduction in performance %	Summer 1.2 met 0.5 clo °C	Summer 1.6 met 0.5 clo °C	Winter 1.2 met 1.0 clo °C	Winter 1.6 met 1.0 clo °C
+2.0	75	12	31.0	30.5	30.9	30.5
+1.5	50	7	29.5	28.4	28.7	27.7
+1.0	25	3	27.8	26.2	26.4	24.6
+0.5	10	1	26.0	24.0	24.0	21.5
0	5	0	24.5	21.8	22.0	18.5
-0.5	10	1	23.0	19.7	19.6	15.2
-1.0	25	3	21.2	17.5	17.1	12.1
-1.5	50	10	19.6	15.3	15.0	
-2.0	75		18.0	13.1	12.5	

PERSONAL CONTROL

Garment	Thermal Insulation	Change of
Description	clo	Operative Temp.
		Κ
Sleeveless vest	0,12	0,8
Thin sweater	0,20	1,3
Light jacket	0,25	1,6
Normal jacket	0,35	2,2

Thermal Comfort- Performance

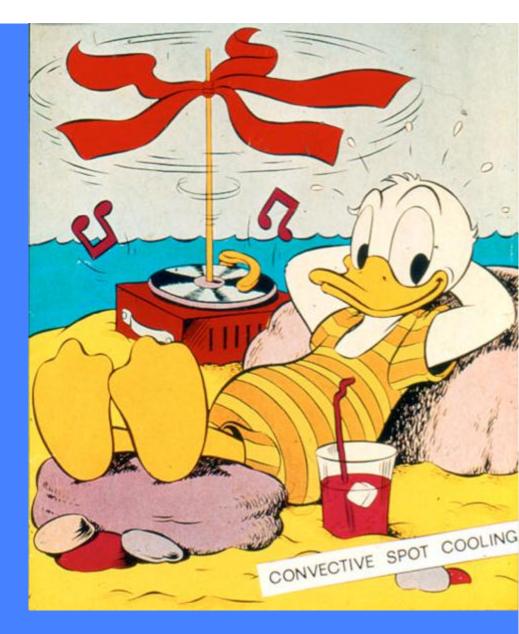
 10 % decrease in dissatisfied will increase the performance with 1,5 %

LOCAL THERMAL DISCOMFORT

- FLOOR SURFACE TEMPERATURE
- VERTICAL AIR TEMPERATURE DIFFERENCE
- DRAUGHT
- RADIANT TEMPERATUR ASYMMETRI

DRAUGHT

- MEAN AIR VELOCITY
- TURBULENCE
- AIR TEMPERATURE



DRAUGHT RATING, DR= (34-ta)(v-0.05)0.62(0.37 v Tu + 3.14)

RADIANT TEMPERATURE ASYMMETRY

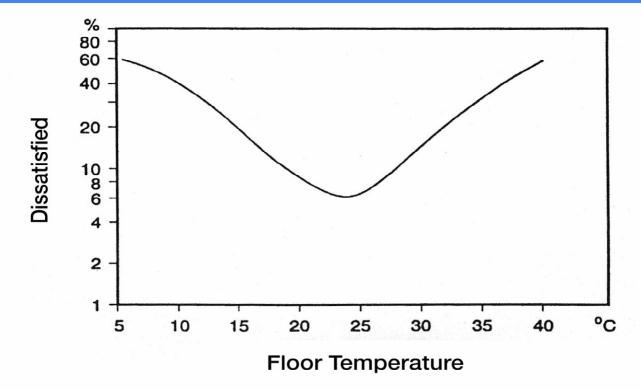
- HEATED CEILING: < 5 °C
- COOLED CEILING: < 14 °C
- WARM WALL:
- COOL WALL:

- < 14 °C
- < 23 °C
- <10 °C

	airt	Vertical emperature ifference
The coal shortage 1917 By occupiing the space near the ceiling the heat will	Category	Vertical air temp. diff. K
be fully utilized- the redundant floor can then be	А	< 2
used for growing potatoes. R. Storm. P.	В	< 3
	С	< 4

FLOOR TEMPERATURE





SEATED/STANDING PERSONS: 19 °C < t_e < 29 °C
BY HIGHER ACTIVITY LEVELS A LOWER FLOOR TEMPERATURE IS ACCEPTABLE

Directive of the European Parliament and of the Council on the energy performance of buildings

The energy performance of buildings should be <u>calculated on</u> <u>the basis of a methodology</u>, which <u>may be differentiated at</u> <u>regional level</u>, that includes, in addition to thermal insulation other factors that play an increasingly important role such as heating and air-conditioning installations, application of renewable energy sources and design of the building.



DIRECTIVE (Art.7) Energy performance certificate

•Member States shall take measures to ensure that for buildings with a total useful floor area over 1 000 m2 occupied by public authorities and by institutions providing public services to a large number of persons and therefore frequently visited by these persons an <u>energy certificate</u>, not older than 10 years, is placed in a prominent place clearly visible to the public.

•The range of recommended and current indoor temperatures and, when appropriate, other relevant climatic factors may also be clearly displayed.

International Centre for Indoor Environment And Energy

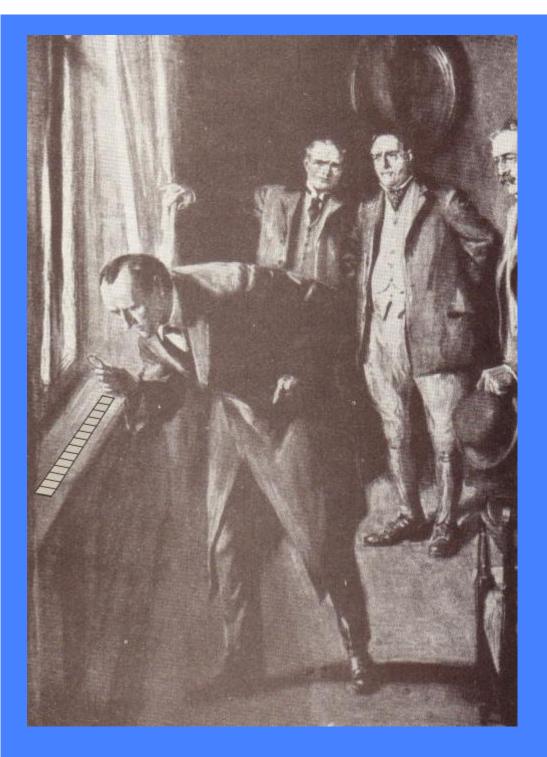


DIRECTIVE (Art.11)

•Moreover, the displaying of officially recommended indoor temperatures, together with the actual measured temperature, should discourage the misuse of heating, air-conditioning and ventilation systems.

•This should contribute to avoiding unnecessary use of energy and to safeguarding comfortable indoor climatic conditions (thermal comfort) in relation to the outside temperature.





INSPECTION

- Boiler and Heating System
- Ventilation System
- Air- (Room) Conditioning System

INSPECTION

- Development of a standard inspection procedure
 - Boilers and heating systems
 - Ventilation systems
 - Cooling systems (Air or Room Conditioning)
- Education of inspectors



DO IT LIKE THE CARE INDUSTRY

INSPECTION

- Use same concept as for cars:
 - Plug in of PC for download of data
 - Build in sensors
 - No fixed inspection intervals (Months or Distance)
 - Calculation of required inspection
 - Self diagnostic

FUTURE NEEDS FOR HVAC SYSTEMS

- More build in sensors for diagnostic and inspection
- Algorithms for calculation of time for service and inspections
- Running calculation of energy consumption
- More separate measurements of electrical energy use
- Feedback to user regarding energy consumption and indoor environment

FUTURE TRENDS

- Integrated design of building and HVAC systems
- Separation of heating-cooling system from ventilation
- Low-temperature heating and high-temperature cooling
- Use of renewable energy sources (heat pumps, ground heat exchangers, geothermal)
- Reduction of peak loads

OPORTUNITIES FOR THE HVAC INDUSTRY

- Earlier involvement in the building process
- More jobs (energy certificates, inspections, service)

CONCLUSIONS

- An optimal indoor environment is the most important requirement for a healthy, comfortable and productive working environment
- For each 10 % increase in dissatisfaction the performance will decrease with ~1.5%
- To reduce energy consumption by decreasing the quality of the indoor environment is a bad investment
- An energy certificate make no sense without a certificate for the indoor environment